

Recent Research Results from the Sandia Advanced Fuels Laboratory



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Research Supported by

***US DOE Office of FreedomCAR and Vehicle Technologies
Program Manager: Kevin Stork***

*Doshisha University
Kyoto, Japan
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Presentation Outline

Overview of Sandia programs and facilities

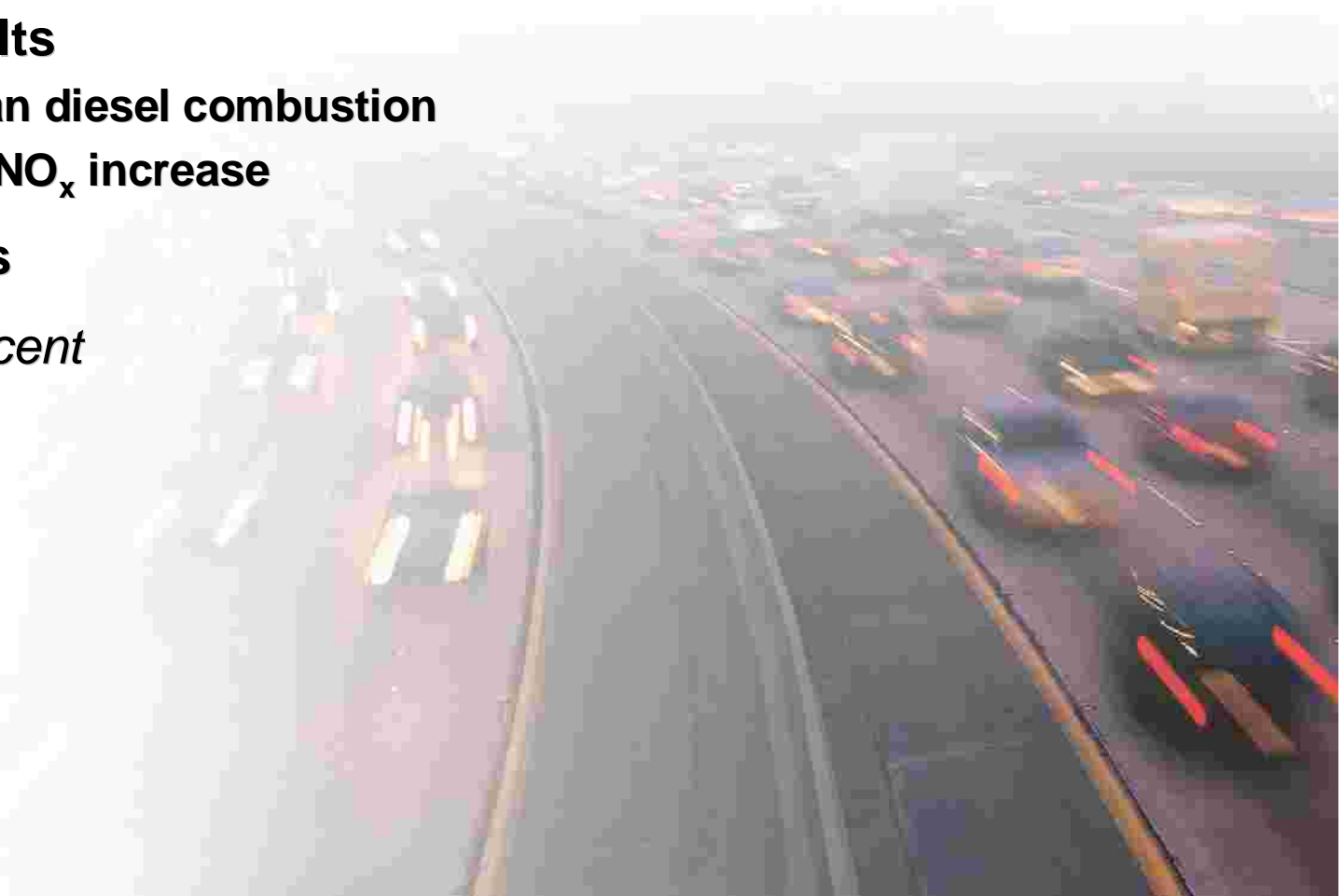
Sandia Advanced Fuels Laboratory

Recent results

- ? **Dilute clean diesel combustion**
- ? **Biodiesel NO_x increase**

Conclusions

*Additional recent
results and
references*



What is Sandia?

A national laboratory funded by the US Dept. of Energy

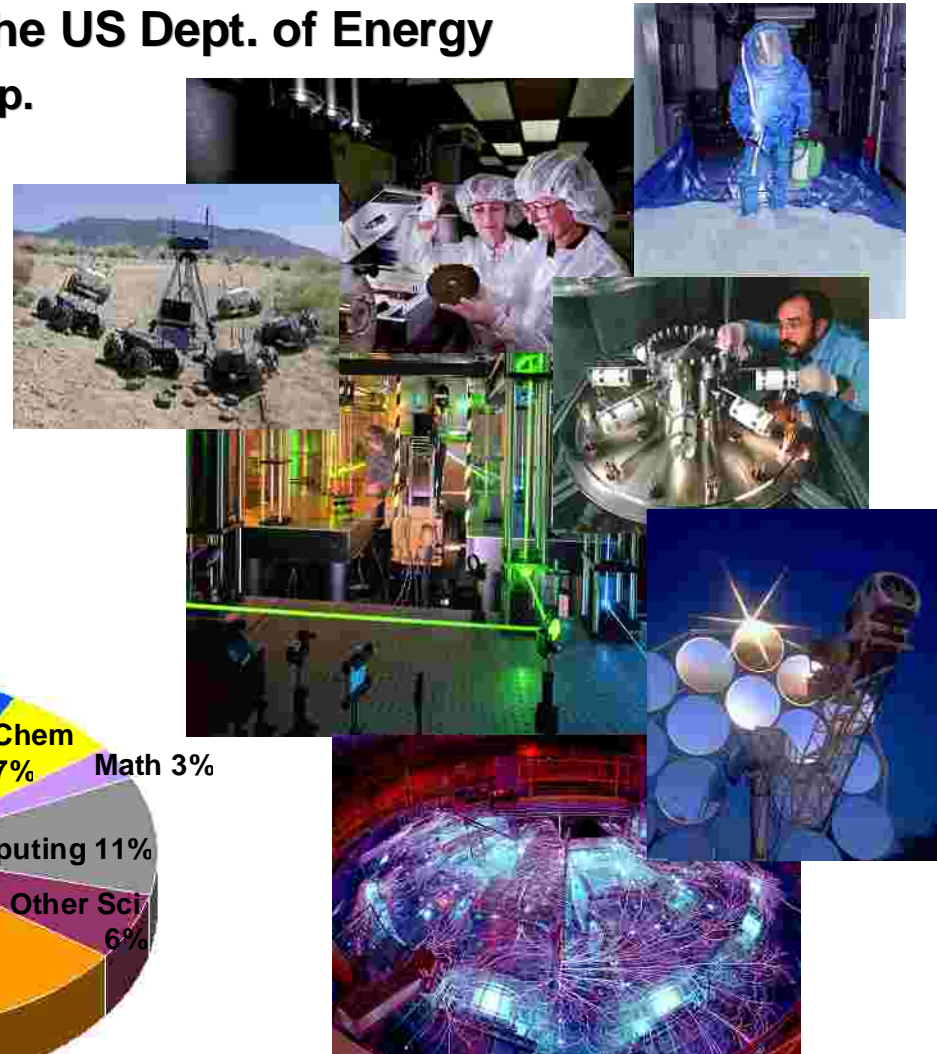
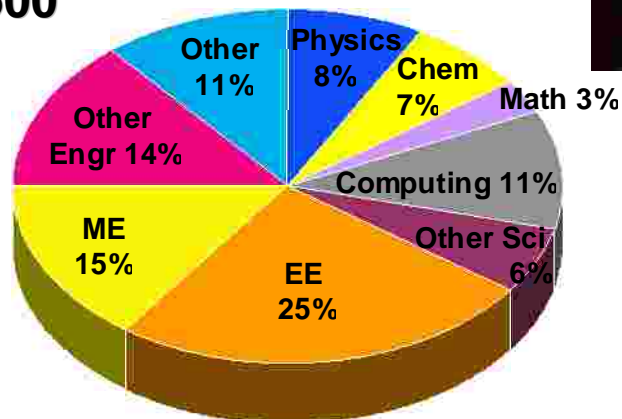
? Operated by Lockheed-Martin Corp.

Technical programs include

- ? Engineering Sciences
- ? Computational and Information Sciences
- ? Materials and Process Science
- ? Pulsed Power Sciences
- ? Microelectronics and Photonics Sciences

Sandia has ~ 7500 employees

- ? 2100 Masters
- ? 1400 Ph.D.
- ? Majority are engineers
- ? 900 in CA



Sandia Has Facilities in Numerous Locations



Albuquerque, New Mexico



**Tonopah Test Range,
Nevada**



WIPP, New Mexico



**Yucca Mountain,
Nevada**



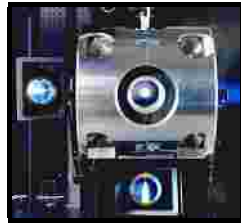
Kauai Test Facility, Hawaii



Livermore, California

Engine-Related Research at the CRF

Engine Combustion Department has 7 laboratories (one principal investigator in each lab, dept. manager is Dennis Siebers)



Constant-Volume Vessel

*Lyle Pickett (PI)
Tim Williams (PD)*



Fuel Effects

*Chuck Mueller (PI)
Glen Martin (PD)
Krishna Lakshminarasimhan (PD)*



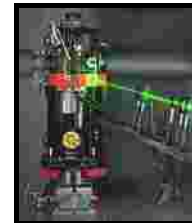
HCCI Fundamentals

*John Dec (PI)
Magnus Sjoberg (LTE)
Wontae Hwang (PD)*



HCCI, Light-Duty

*Richard Steeper (PI)
Russ Fitzgerald (PD)*



HECC, Heavy-Duty Diesel

Mark Musculus (PI)



HECC, Light-Duty Diesel

*Paul Miles (PI)
Will Colban (PD)
Isaac Ekoto (PD)*



HECC, Hydrogen

Sebastian Kaiser (PI)

Sandia Fuels Project Vision

High-efficiency, clean combustion (**HECC**) using advanced and/or non-petroleum fuels:

- Robust operation
- Acceptable heat release
- High power density

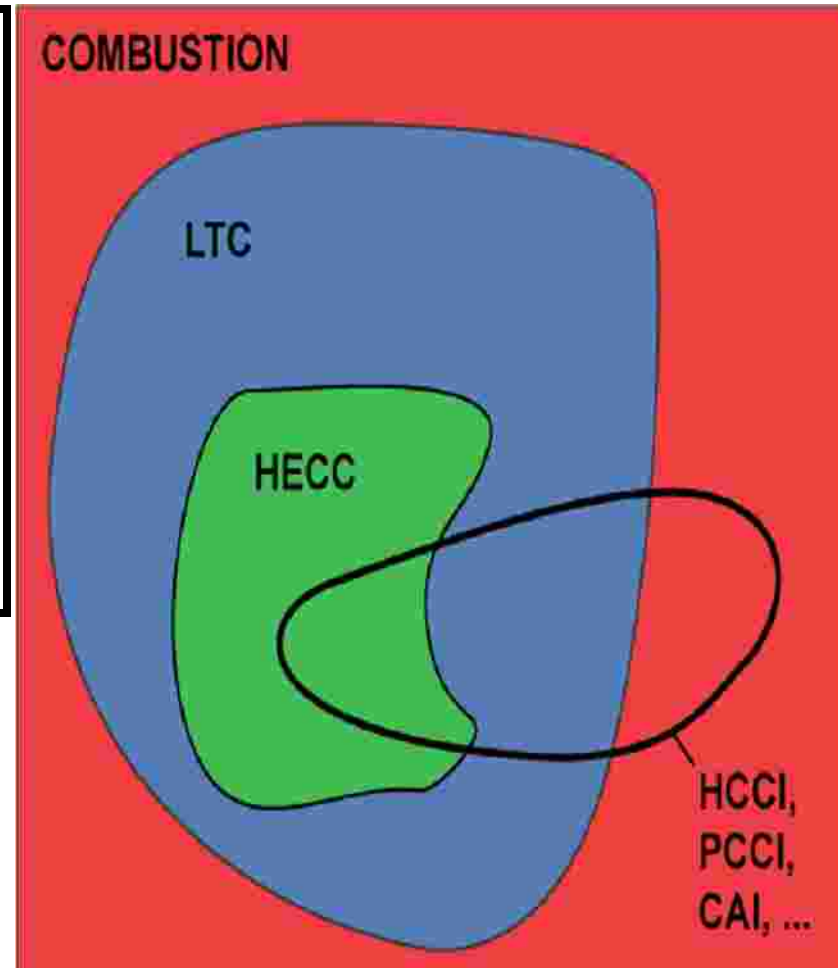
HECC = High-Efficiency, Clean Combustion

- *Efficiency similar to conventional diesel*
- *US 2010 heavy-duty regulations achieved with oxidation catalyst*

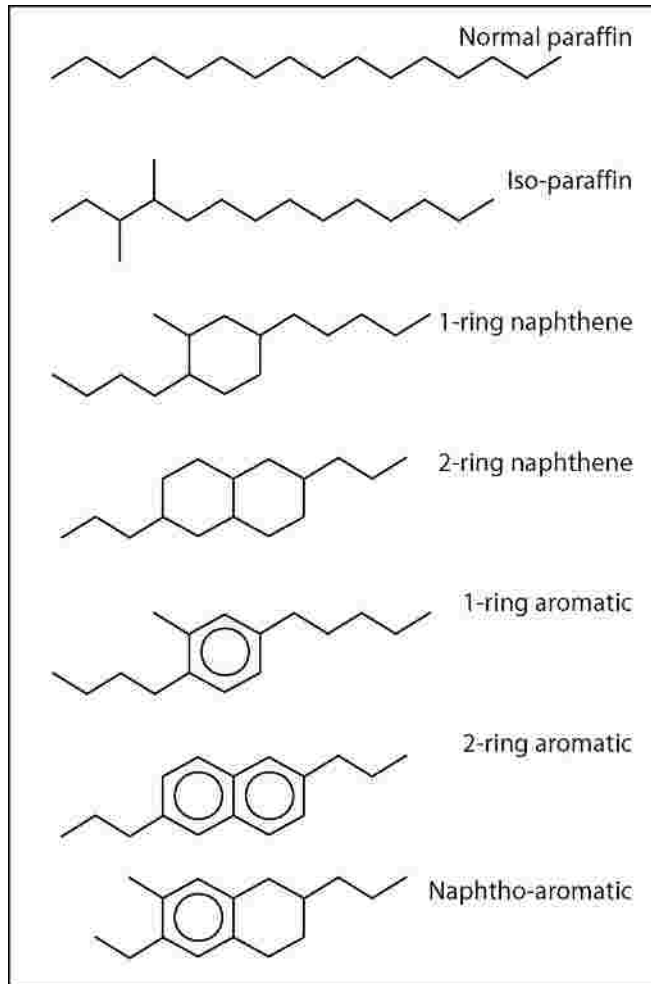
LTC = Low-Temperature Combustion

- *No constraints on efficiency*
- *Peak T low enough to minimize NO_x*

HCCI = Homogeneous Charge Compression Ignition



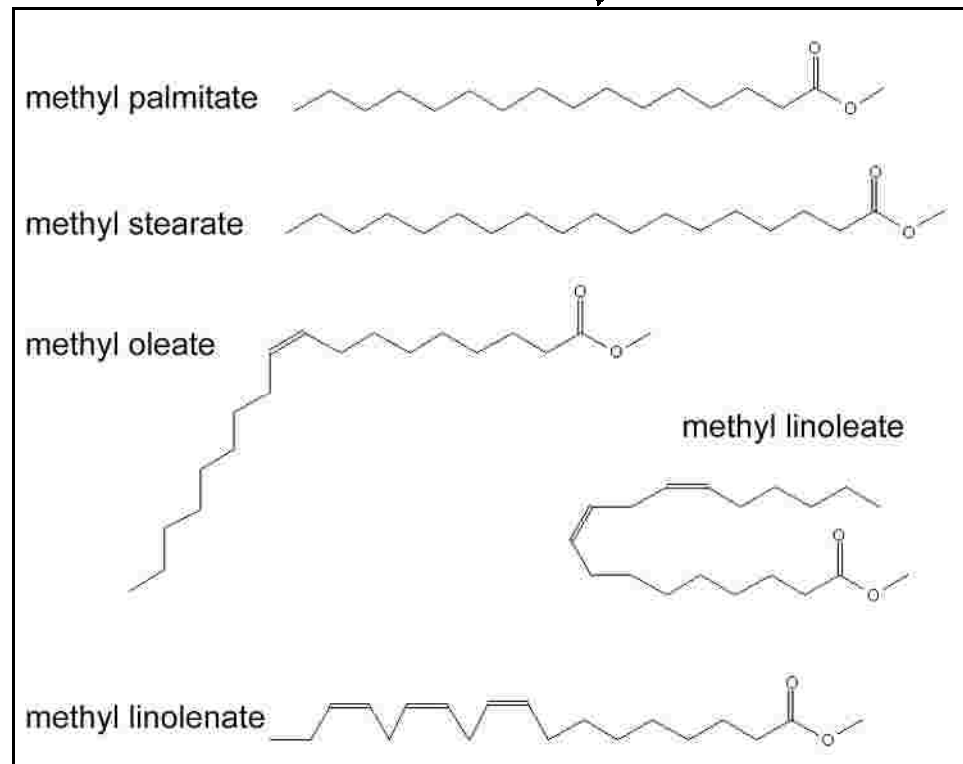
New Fuels Can Be Very Different from Conventional Fuels, and Burn Differently



from Farrell et al., SAE 2007-01-0201

Conventional diesel

Biodiesel (soy methyl esters)

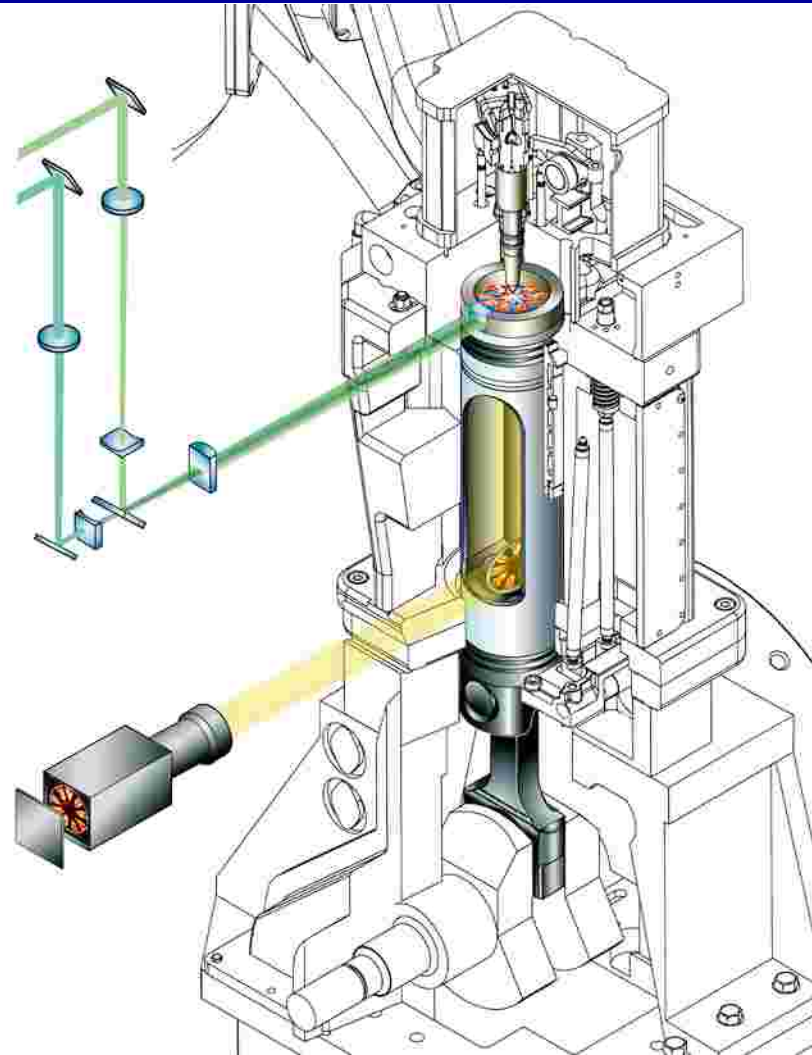


from Mueller, CRF News, May/June 2006

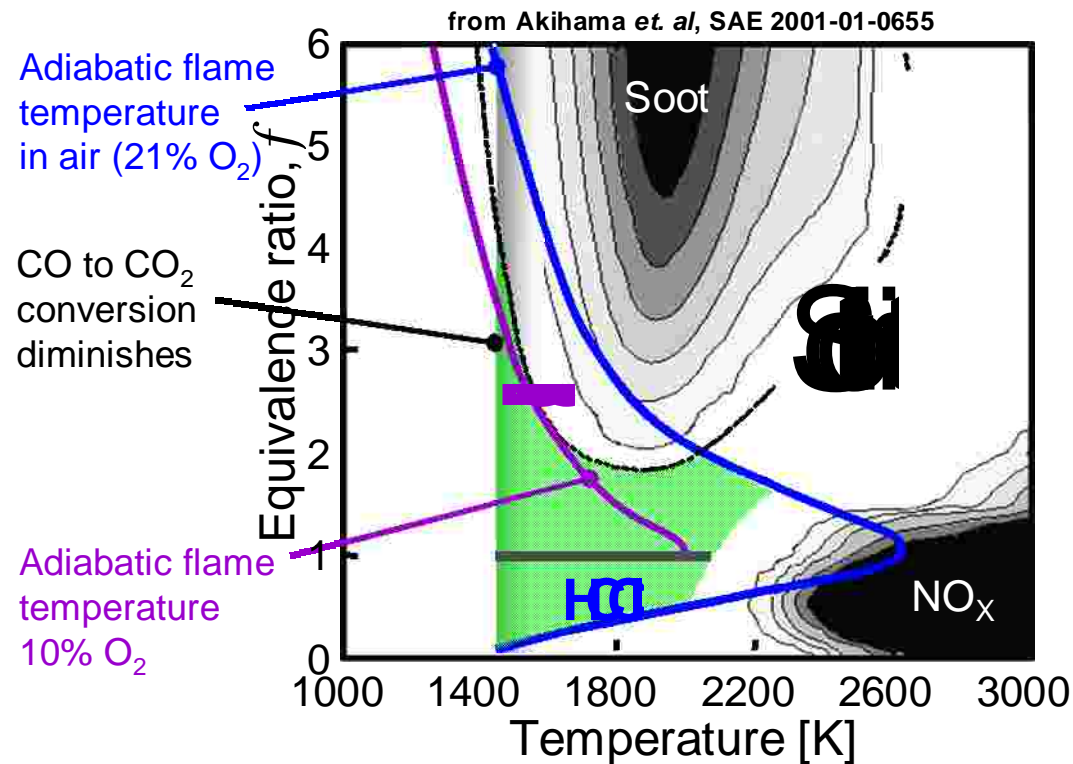
Optical Engine Specifications and Schematic

Research engine	1-cyl. Cat 3176
Cycle	4-stroke CIDI
Valves per cylinder	4
Bore	125 mm
Stroke	140 mm
Conn. rod length	225 mm
Conn. rod offset	None
Piston bowl diameter	90 mm
Piston bowl depth	16.4 mm
Squish height	1.5 mm
Swirl ratio	0.59
Displacement per cyl.	1.72 liters
Compression ratio	11.3:1
Simulated compr. ratio	16.0:1

Quartz windows in piston and upper periphery of cylinder liner enable optical access



Dilute Clean Diesel Combustion (DCDC) Using Oxygenated and Emerging Fuels



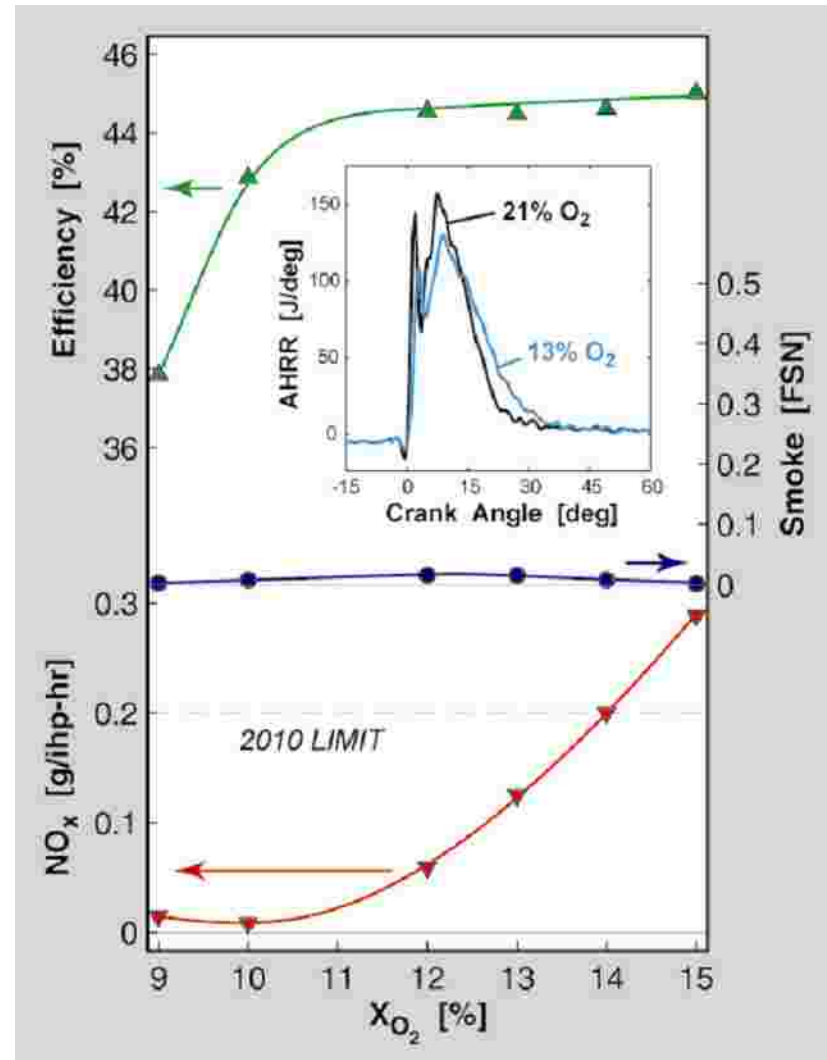
DCDC Simultaneously Achieves Low Emissions and High Efficiency

Operating conditions

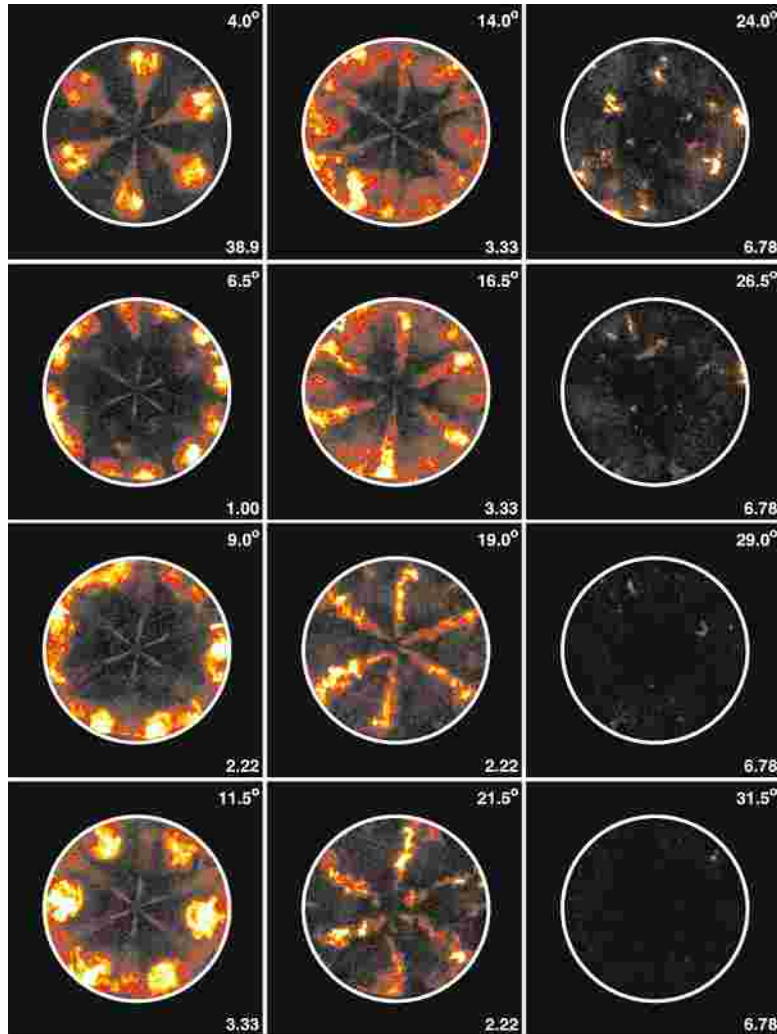
- ? Diethylene glycol diethyl ether (DGE) fuel
- ? 1200 rpm, 7-bar IMEP
- ? Steady-state operation
- ? -9 ° C, 1.4 bar intake
- ? EGR simulated by N₂ dilution

High-load operation also demonstrated in optical engine

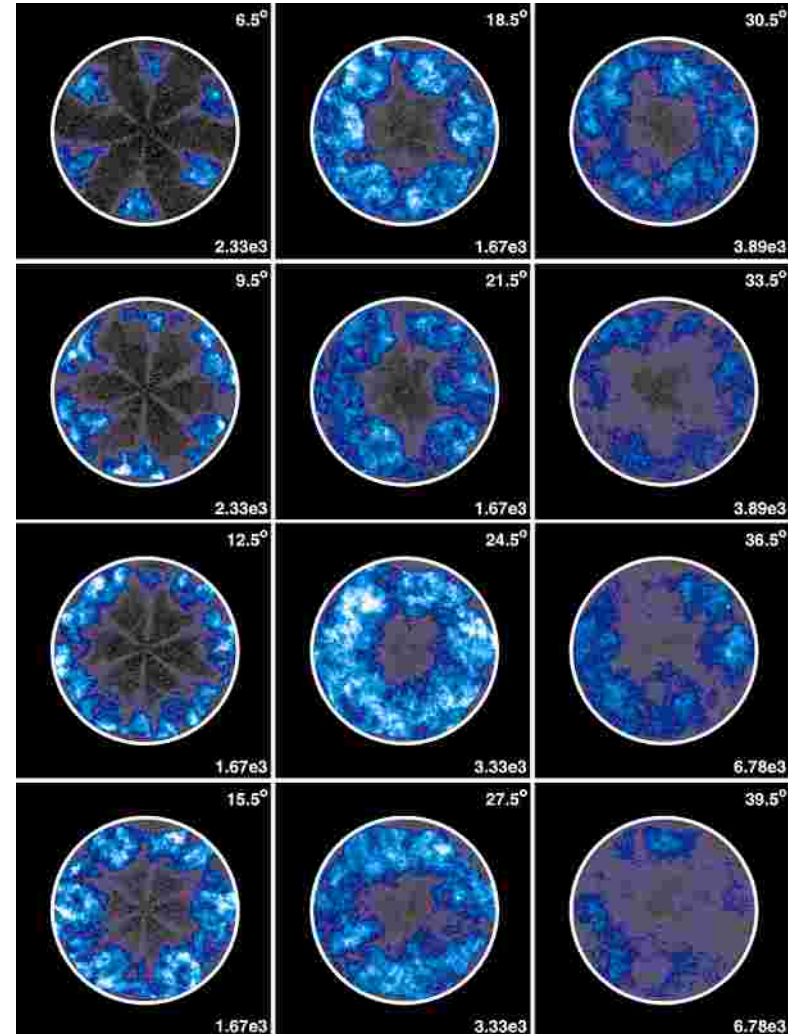
- ? 18-bar IMEP (2/3 load)
- ? 0.09 g/ihp-hr NO_x
- ? 0.26 FSN smoke
- ? Load and efficiency limited by peak cylinder pressure capability of optical engine



Natural Luminosity Imaging of Undiluted and Highly Dilute Combustion (DGE Fuel)



Undiluted



9% Intake-O₂ Mole Fraction

More-Conventional Fuels and Operating Conditions

Fuels

- ? **B100 = neat soy biodiesel (Peter Cremer Nexsol BD-0100)**
- ? **CN80 = 80-cetane PRF blend, 76.5 vol% *n*-hexadecane + balance heptamethylnonane**
- ? **D2 = Phillips #2 diesel reference fuel**

Operating conditions

- ? **1200 rpm, steady-state**
- ? **6.7 bar IMEP load**
- ? **Peak injection pressure = 1420 bar, 6 x 0.163 mm x 140 ° nozzle**
- ? **Start of injection £ 4 ° BTDC**
- ? **Start of combustion between 0 ° (TDC) and +0.5 ° ATDC**
- ? **Simulated intake conditions: 53 ° C, 1.8 bar (abs.)**
- ? **960 K, 31 kg/m³ @ TDC (motored)**
- ? **EGR simulated using nitrogen dilution**

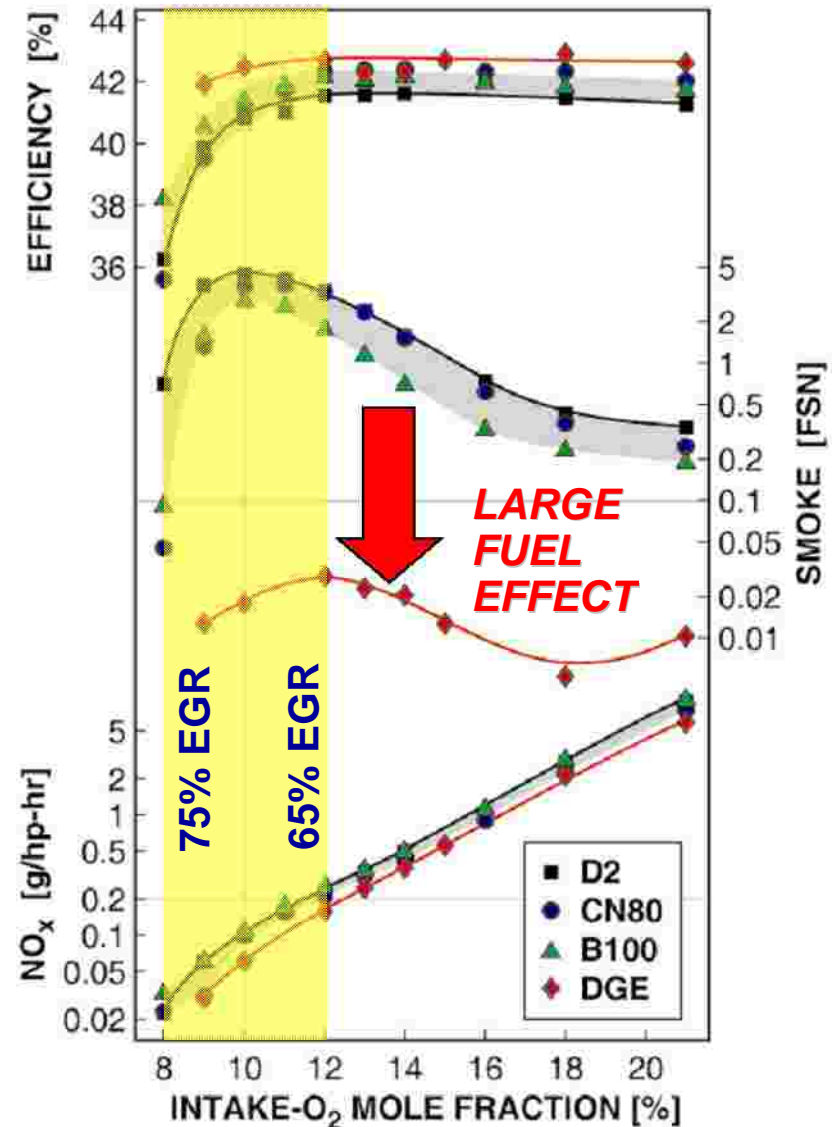
To What Extent Is DCDC Possible with Conventional and Emerging Fuels? [1,2]

Answer: With near-term fuels, smoke emissions become ~50X too large at required EGR levels

- ? Efficiency drops before compliant smoke emissions obtained
- ? Highly oxygenated fuel can remove this barrier

Very high levels of cooled EGR required for NO_x compliance

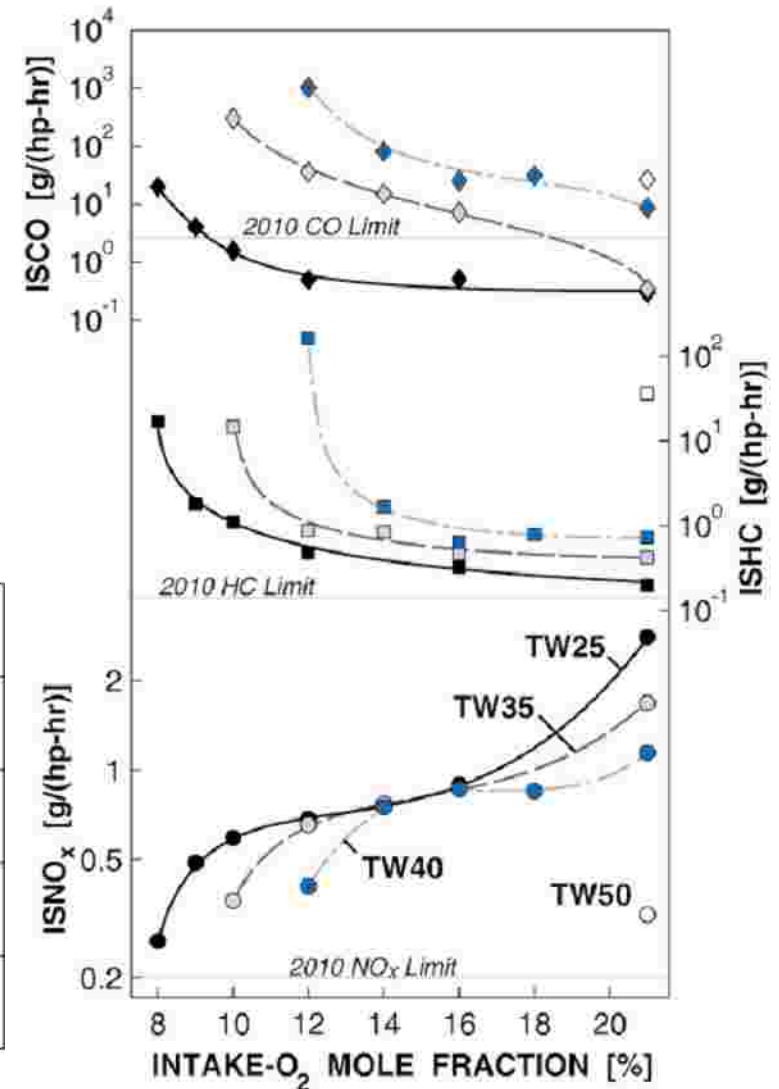
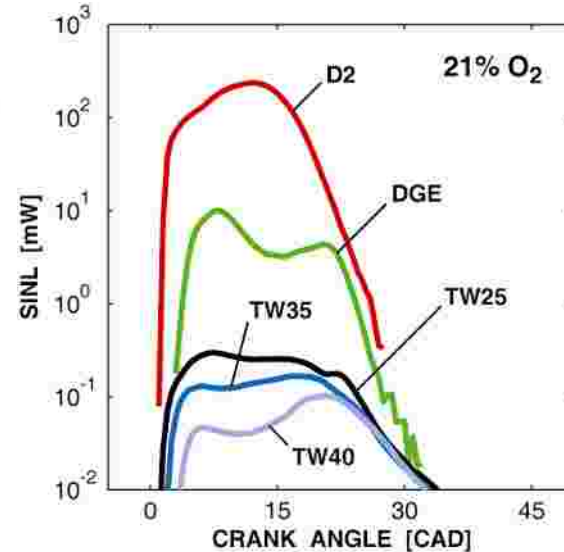
- ? Is there a better way to introduce diluent?



Are Fuel-Water Mixtures a Viable Alternative to EGR for NO_x Control? [3]

Answer: Yes, potentially

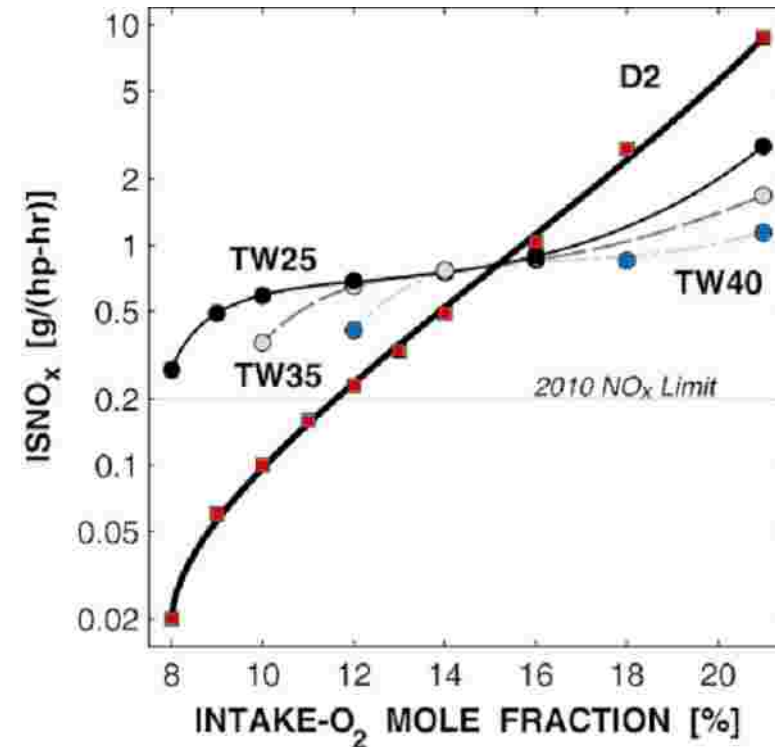
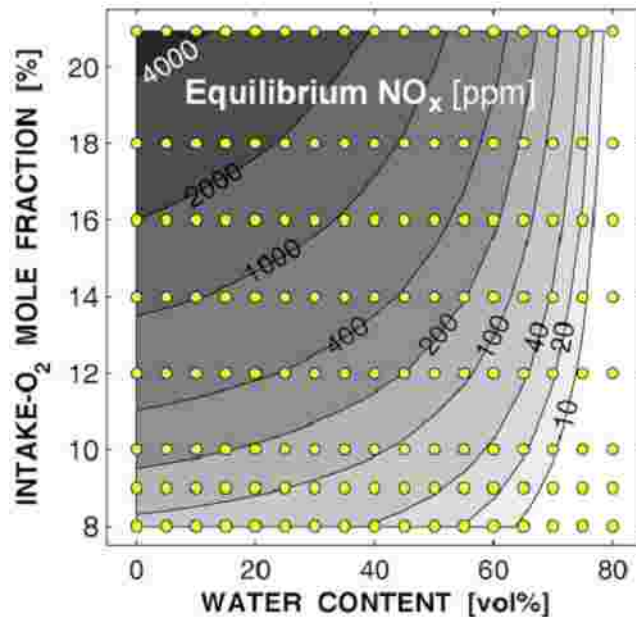
- ? Studied blends of tri-propylene glycol methyl ether with 25 to 50 vol% water
- ? Stable blends pass corrosivity tests for ferrous metals and copper
- ? Ignition delay \gg 40-50 cetane #2 diesel
- ? Can lower NO_x by $\sim 10x$ without using EGR (relative to #2 diesel)
- ? HC and CO compliance possible with oxidation catalyst
- ? Exceedingly low in-cyl. soot and engine-out smoke



Are There Limitations to the Benefits of Fuel-Water Mixtures? [3]

Answer: Yes.

- ? Incomplete combustion when > 40 vol% water
- ? Even 10x lower NO_x isn't enough for 2010 compliance need EGR or NO_x aftertreatment
- ? NO_x doesn't decrease as rapidly as expected with simulated EGR



- ? Equilibrium calc's corroborate that NO_x doesn't decrease quickly with EGR @ high water content
- ? Even so, fuel-water mixtures could be part of a successful strategy

Biodiesel NO_x [4]

NO_x emissions increase by ~1% for every 10 vol% biodiesel blended into diesel fuel.

Why?

Possible Reasons for Biodiesel NO_x Increase

**Increased residence time at higher in-cylinder temperatures
higher NO_x**

- 1. Higher bulk modulus earlier injection earlier combustion**
- 2. Shorter ignition delay earlier combustion**
- 3. Larger premixed-burn heat release**
- 4. Higher adiabatic flame temperature**
- 5. Less in-cylinder soot less radiative heat transfer higher actual flame temperatures**
- 6. Mixture-stoichiometry effects (thermal, chemical-kinetic)**
- 7. Others...?**

Experiment Design

Assess mechanisms 1-4 by comparing biodiesel results to those using a hydrocarbon reference fuel with same:

- 1. Injection timing** to remove bulk-modulus effect
- 2. Start of combustion** to remove combustion-phasing effect
- 3. Ignition delay** to remove premixed-burn magnitude effect
- 4. Adiabatic flame temperature**

If above matching is accomplished

and differences in one (or more) of these is primary cause of biodiesel NO_x increase

then biodiesel NO_x increase should vanish.

Fuels and Operating Conditions

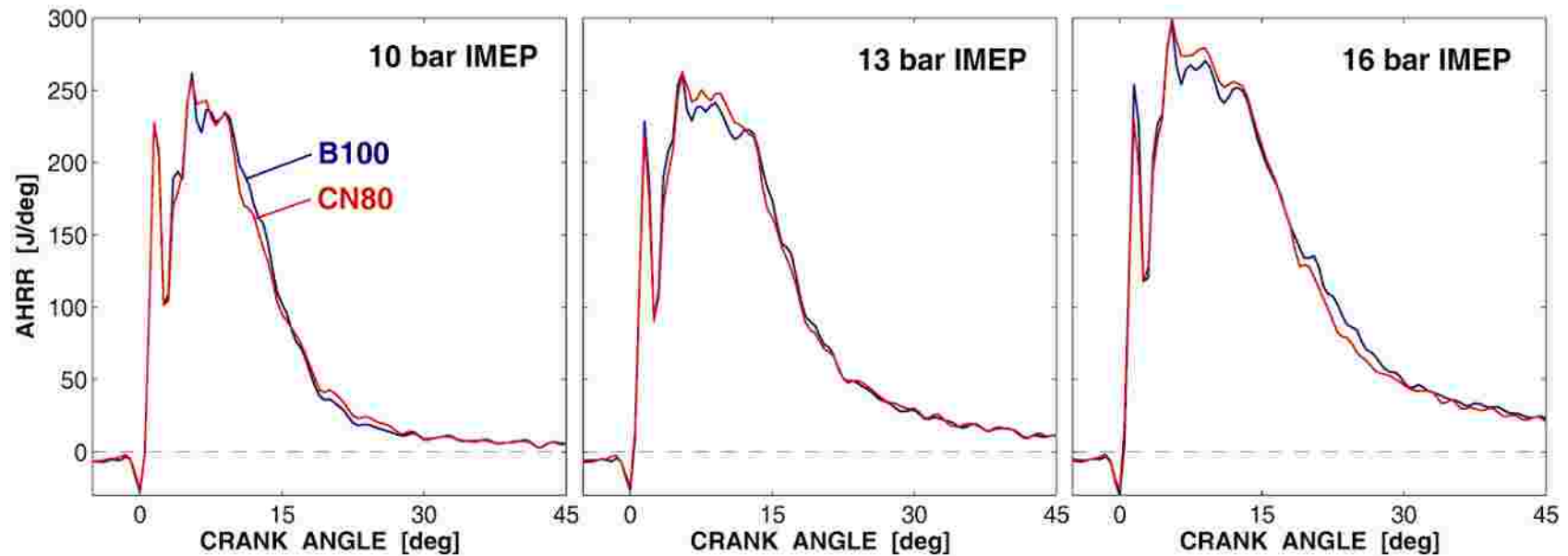
Fuels

- ? **B100 = neat soy biodiesel (Peter Cremer Nexsol BD-0100)**
- ? **CN50 = 50-cetane blend of diesel primary reference fuels (PRFs), 41.2 vol% *n*-hexadecane + balance 2,2,4,4,6,8,8-heptamethylnonane**
- ? **CN80 = 80-cetane PRF blend, 76.5 vol% *n*-hexadecane + balance heptamethylnonane**
- ? **CN100 = 100-cetane PRF (*i.e.*, neat *n*-hexadecane)**

Operating conditions

- ? **Selected to maximize NO_x differences between B100 and PRFs**
- ? **800 rpm, steady-state, 0% EGR, conventional injection timing, start of combustion between 0 ° (TDC) and +0.5 ° ATDC**
- ? **Loads from 10 to 16 bar gross IMEP in 1-bar increments**
- ? **Peak injection pressure = 1420 bar, 6 x 0.163 mm x 140 ° nozzle**
- ? **Simulated intake conditions 69 ° C, 1.43 bar for 16:1 CR engine**
- ? ³ **4 repeats at each operating condition**

Similar AHRR Curves for B100 and CN80

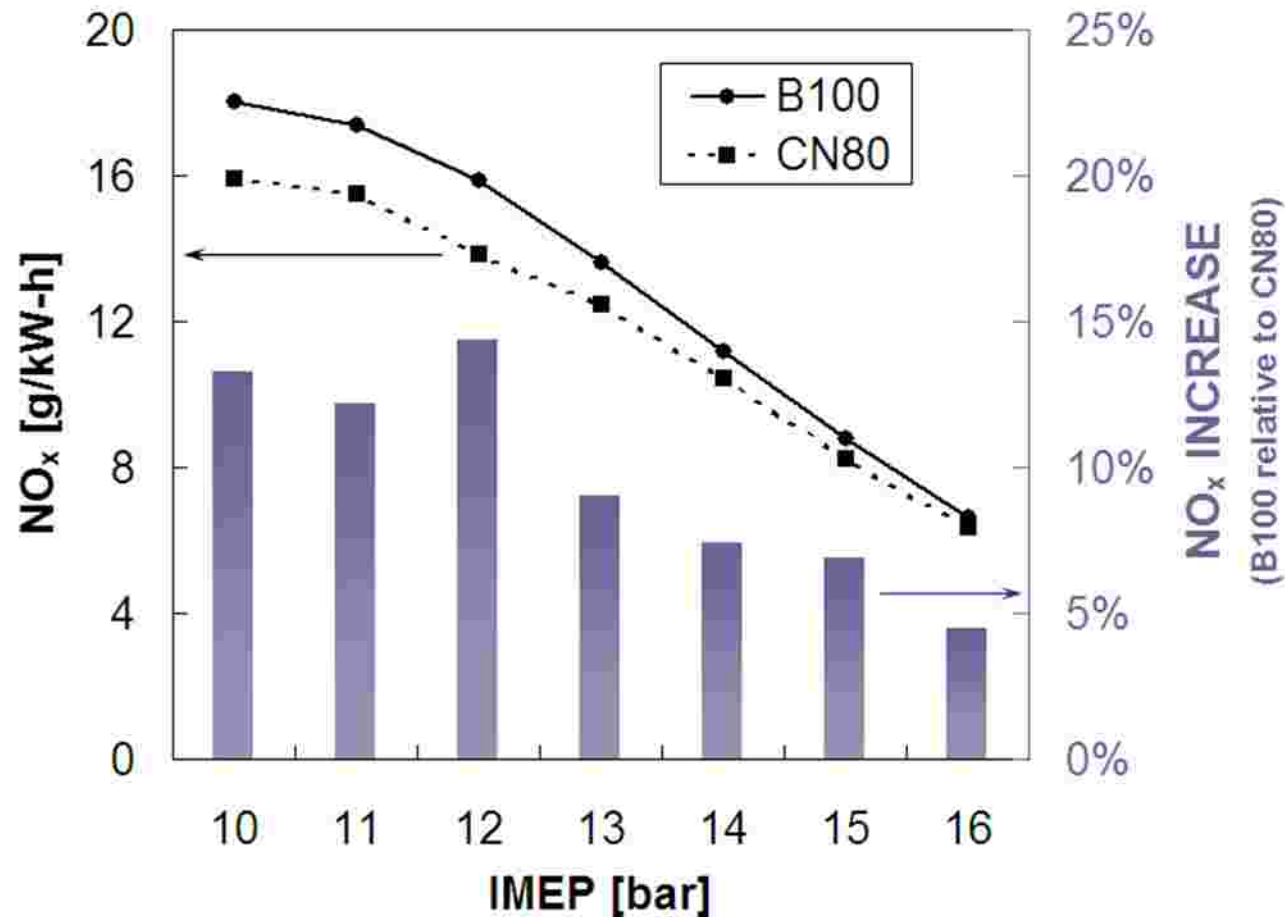


Start of injection, start of combustion, and premixed-burn magnitude well matched across load range

Mixing-controlled heat release also well matched

**? 15% lower LHV of B100 is mostly offset by 12% higher density
injection duration approx. same as for CN80**

NO_x Is 10.5% Higher for B100 Than CN80



Factors other than ignition delay, start of combustion and premixed-burn magnitude contribute to increased biodiesel NO_x

Conclusions

Dramatic fuel changes can enable mixing-controlled HECC, avoiding common problems of more-premixed LTC strategies

- ? **No problems with ignition-timing control, light-load misfire, or high-load knock and NO_x**

Nevertheless, significant technical advancements are required to enable *practical* mixing-controlled HECC

- ? **Mixture preparation and fuel must be optimized together to avoid:**
 - † Excessive EGR requirements for NO_x control
 - † High smoke emissions with current and emerging fuels

Current hypotheses are inadequate to explain the NO_x increase when fueling with biodiesel (relative to a diesel PRF blend)

- ? **Changes in start of injection, start of combustion, premixed-burn magnitude, and adiabatic flame temperature are not controlling factors over a range of load conditions**
- ? **Differences in radiant heat transfer and/or mixture stoichiometry could play roles (investigation underway)**

Acknowledgments

Dr. Glen C. Martin – *post-doc*

Caterpillar Inc. – *hdwe. support, guidance*

Dr. Ansis Upatnieks – *former post-doc*

Prof. A.S. (Ed) Cheng – *collaborator*

Dr. Randy L. Vander Wal – *collaborator*



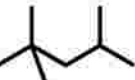
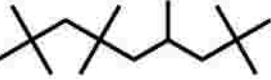
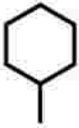
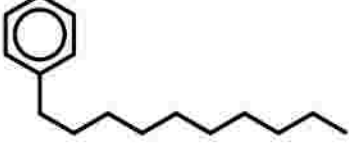
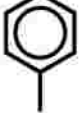
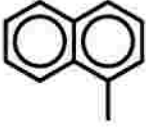
**Advanced Engine Combustion,
Diesel Surrogates – *working groups***

Questions



What Compounds Should Be Used to Create a Diesel Surrogate for Kinetics Studies? [5]

Answer: Diesel Surrogates Working Group has selected...

Near-Term	Longer-Term
<p><i>n</i>-decane (C₁₀H₂₂) </p>	<p><i>n</i>-hexadecane (C₁₆H₃₄) </p>
<p>iso-octane (C₈H₁₈) </p>	<p>heptamethylnonane (C₁₆H₃₄) </p>
<p>methyl cyclohexane (C₇H₁₄) </p>	<p><i>n</i>-decylbenzene (C₁₆H₂₆) </p>
<p>toluene (C₇H₈) </p>	<p>1-methylnaphthalene (C₁₁H₁₀) </p>

- ? Compounds can be blended to match characteristics of real diesel: ignition delay, molecular structures, C/H ratio, volatility, ...
- ? Compounds readily obtainable for experimental research efforts
- ? Detailed kinetic mechanisms already exist or can be developed

How to Quantify Mixture Stoichiometry Once Reactions Have Begun? [6]

Answer: The oxygen equivalence ratio, f_{Ω}

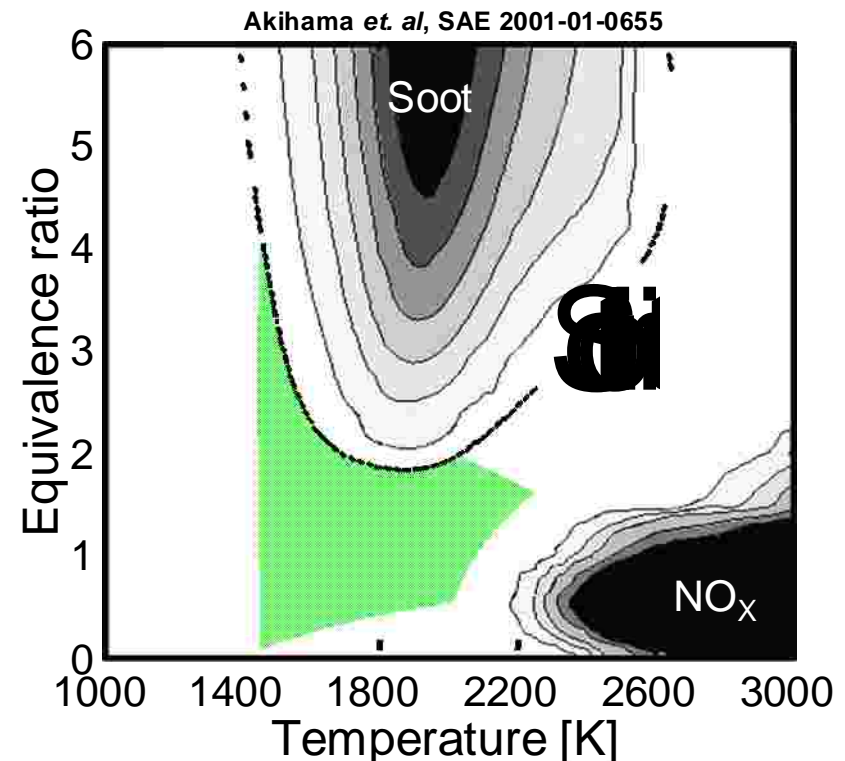
$$f_{\Omega} \equiv \frac{2n_C + \frac{1}{2}n_H}{n_O}, \text{ neglecting atoms bound in CO}_2 \text{ and H}_2\text{O}$$

Important for tracking reaction progress in (f, T) space

Important when oxygenated fuels are used (e.g., biodiesel, ethanol, DME, ...)

Same as traditional f definition before reactions have begun

- ? As long as fuel not oxygenated
- ? General relationship between f and f_{Ω} provided



How to Quantify Degree of Achievement of Many Simultaneous Operational Targets? [2]

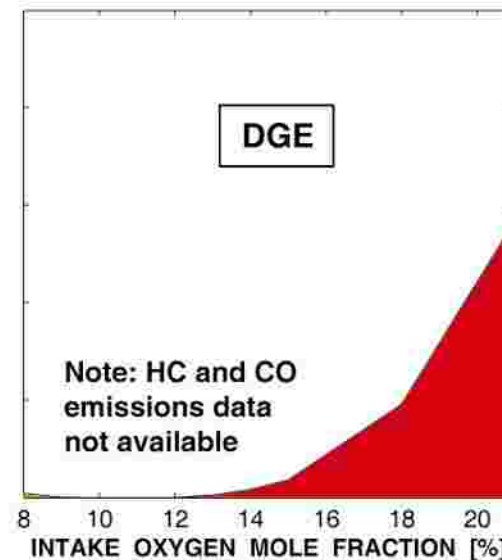
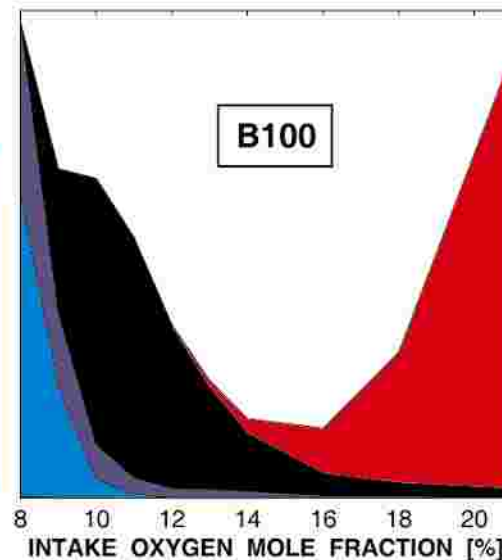
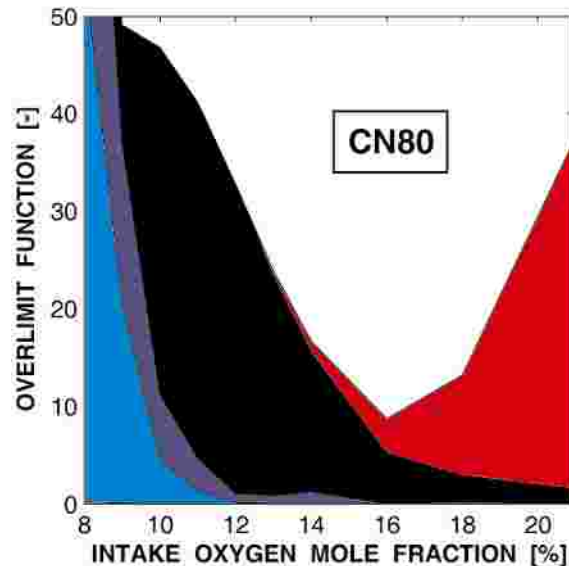
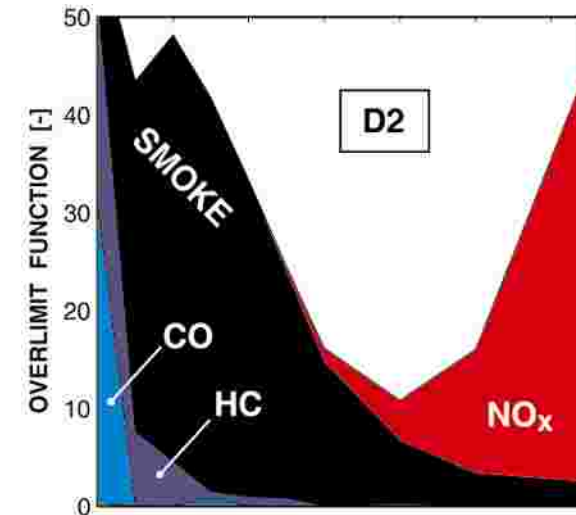
Answer: Overlimit function succinct evaluation of constrained systems

$$F \equiv \sum_i \max\left(0, \frac{x_i}{x_i^*} - 1\right) \quad \text{where}$$

$x_i \equiv i^{\text{th}}$ constrained output parameter

$x_i^* \equiv$ constraint on i^{th} parameter

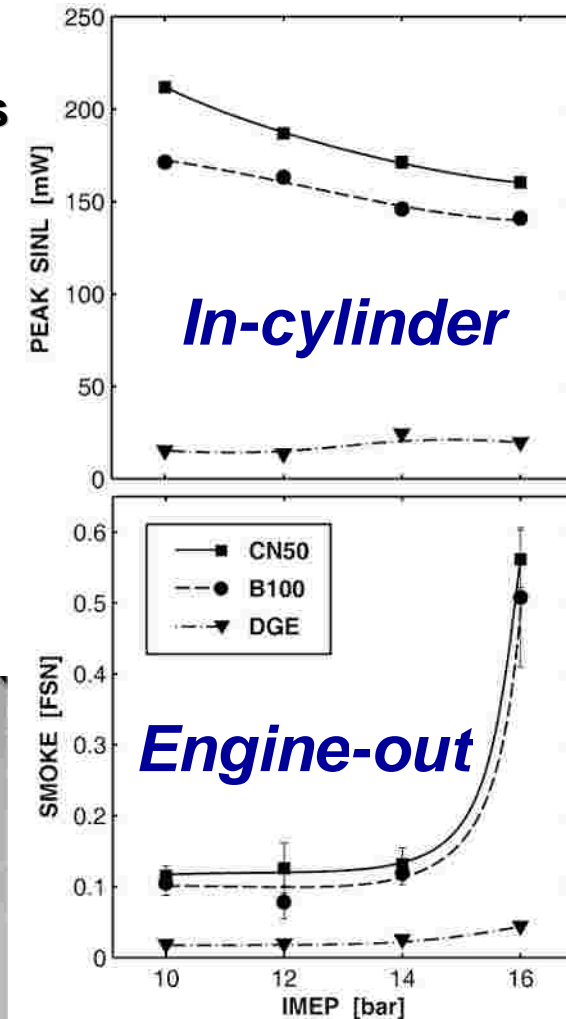
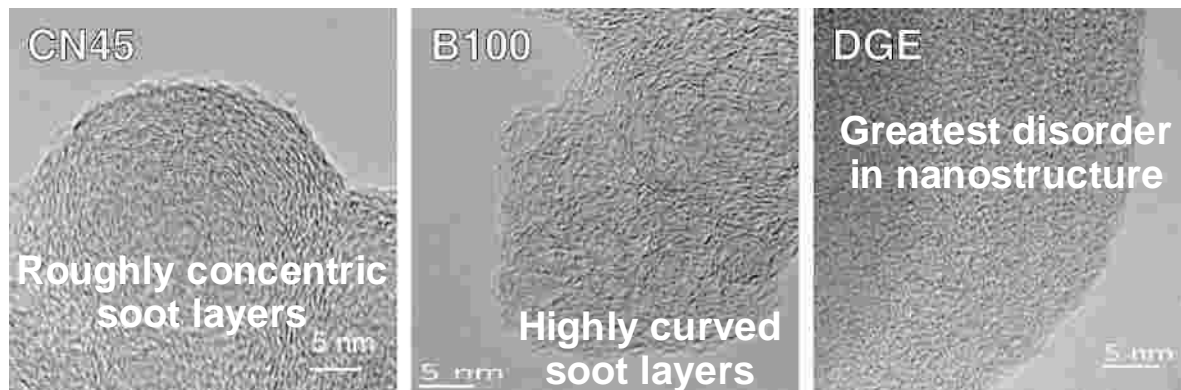
$i \equiv$ index over constrained parameters



Could Fuel Effects on Soot Nanostructure Affect Emissions? [7]

Answer: Yes.

- ? Soot with less-ordered nanostructure oxidizes up to 5X faster
- ? Soot produced by different fuels has different nanostructure (similar operating conditions)
 - † Hydrocarbon ref. fuel highly ordered soot
 - † Neat biodiesel less order in nanostructure
 - † DGE greatest disorder in nanostructure
- ? DGE soot has shortest fringe lengths and largest tortuosity enhanced oxidation rate



References

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2. Cheng, A.S., Upatnieks, A., and Mueller, C.J., "Investigation of Fuel Effects on Dilute, Mixing-Controlled Combustion in an Optical Direct-Injection Diesel Engine," *Energy and Fuels*, in press 2007.
3. Mueller, C.J. and Upatnieks, A., "Operational Characteristics of Oxygenate-Water Fuel Blends Studied in an Optical DI Diesel Engine with Simulated Exhaust Gas Recirculation," SAE Technical Paper 2007-01-2017.
4. Cheng, A.S., Upatnieks, A., and Mueller, C.J., "Investigation of the Impact of Biodiesel Fueling on NO_x Emissions Using an Optical Direct-Injection Diesel Engine," *Int. J. Engine Res.* **7**:297:318, 2006.
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7. Vander Wal, R.L. and Mueller, C.J., "Initial Investigation of Effects of Fuel Oxygenation on Nanostructure of Soot from a Direct-Injection Diesel Engine," *Energy and Fuels* **20**:2364-2369, 2006.