

## **Recent Trends in Diesel Combustion Research in the US and Research Activities at the ERC**



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**Phil and Jean Myers Professor**  
**Engine Research Center**

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## **Quick Assessment**

- **Combustion of fossil fuels is the dominate power producing process in the US**
  - Or, fossil fuels are the dominate energy carrier used in the US
- **Transportation is a significant proportion of the total power generation within the US**
- **The US is realizing the importance of reducing fuel consumption in the mobility sector**
  - Trucking industry has always had this constraint
  - Automotive sector is realizing it also

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## What about CO<sub>2</sub> Emissions?



## Life Cycle Energy Costs & Wells to Wheels?

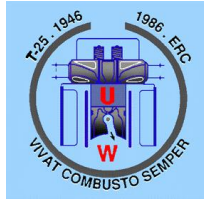
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## Bio-fuels?

- **Bio fuels are oxygenated HC's**
  - They will not be as high in specific energy or energy density as gasoline or diesel. But they will be better than H<sub>2</sub> or batteries
- **With Bio fuels you are “recycling” the carbon emissions**
- **Bio fuels are a complicated topic**
  - Need to do accurate accounting of all growing, harvesting and processing energy and emissions
  - It appears that Bio fuels can be a contributor to the fuel pool, but are not THE solution

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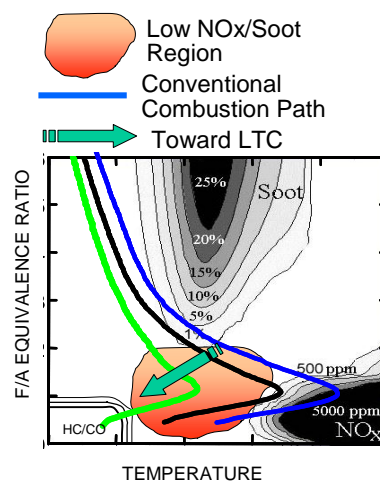
# Engine Combustion and System Integration Research



The current emphasis in the US

## Low-Temperature Combustion (LTC)

- **Critical Issues:**
  - Practical “windows” can be identified:
    - $T < 2100$  K to keep  $\text{NO}_x$  from forming
    - $T > 1500$  K to generate sufficient OH to complete oxidation of CO and HC
  - Exhaust temperature are low with LTC
    - Catalytic clean up of the exhaust may be difficult
  - The soot and  $\text{NO}_x$  islands were determined by static calculations of,  $T = 1.0$  ms,  $P = 6$  MPa and  $\text{EGR} = 0\%$  - In reality they move!



Concept was originally proposed by Kamimoto, SAE 880423

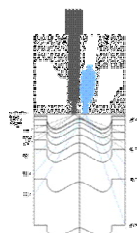
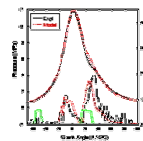
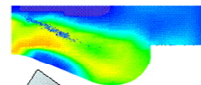
# Low Temperature Combustion

- **Key to LTC:**
  - Achieve appropriate mixing of fuel and oxidizer prior to the ignition chemistry progressing to auto-ignition, which is to occur within some designated time during the cycle
- **Many different scenarios have been proposed for achieving this**
  - HCCI, PCCI, MK, DCDC, CAI, ...
- **These different scenarios are really different approaches to mold the engine operation around the fuel's physical and auto-ignition characteristics**

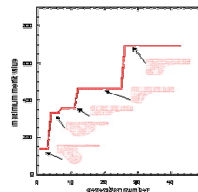
# Combustion Optimization and New Combustion Regimes

**Task 1: Fundamental understanding of LTC-D and advanced model development**

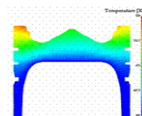
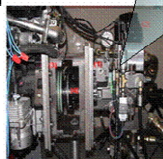
**Task 2: Experimental investigation of combustion control concepts**



**Task 3: Application of models for Optimization of combustion & emissions**

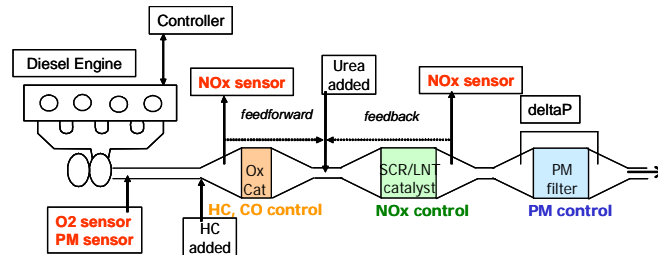


**Task 5: Transient engine control with mixed-mode combustion**



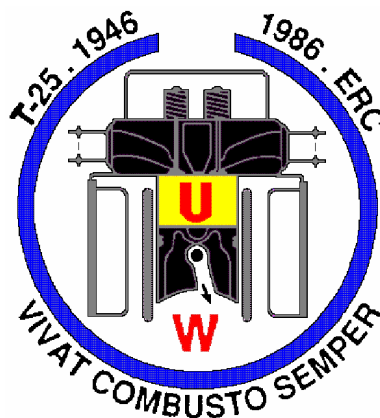
**Task 4: Impact of heat transfer and spray impingement on LTC-D combustion**

## Powertrain Integration and Total System Optimization



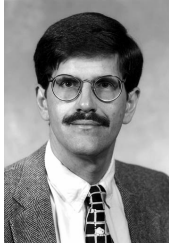
- The aftertreatment system, which is passive, will dictate the engine operating conditions
- Engine needs to supply the exhaust thermodynamic state and composition that is needed for optimum aftertreatment performance at that instant.
- Detailed fundamental understanding of each sub-system will be required

## The Engine Research Center



<http://www.erc.wisc.edu/>

## ERC Faculty



**Mike Corradini**



**Pat Farrell**



**Dave Foster**



**Jaal Ghandhi**



**Rolf Reitz**



**Chris Rutland**



**Scott Sanders**

Plus many active  
collaborations  
with other faculty



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## ERC Research Projects



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# Diesel HCCI/LTC

**Dave Foster, Rolf Reitz, Chris  
Rutland, Youngchul Ra, Manuel  
Gonzalez, Roger Krieger, Richard  
Opat, Ryan Butts**

GM-ERC-CRL, DOE, BP-Amoco



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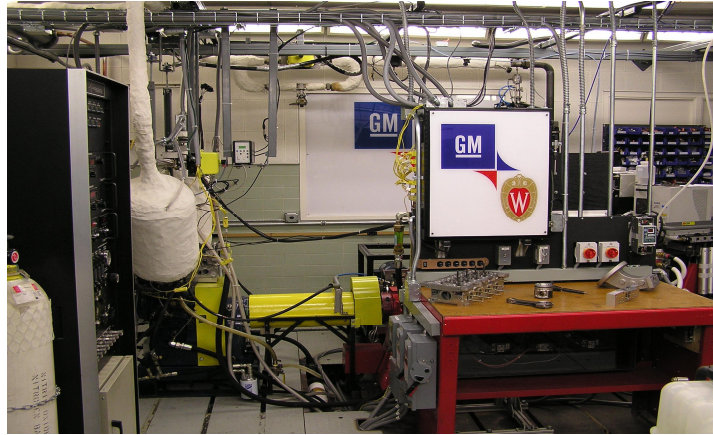
## Diesel LTC (HCCI) Single-cylinder Lab

- **Objective:** *To explore injection, in-cylinder conditions and fuel effects on mixing and combustion regimes in a High Speed DI LTC-D engine*
- **Approach:** To incorporate a study fuel volatility and cetane number effects (focus on properties similar to those of naphtha's, light kerosene and cyclo-alkanes) on LTC-D, knowing the non-linear coupling between the ignition kinetics (both two stage and single stage), the temperature and pressure time histories of the cylinder gases and the mixing processes occurring within the cylinder



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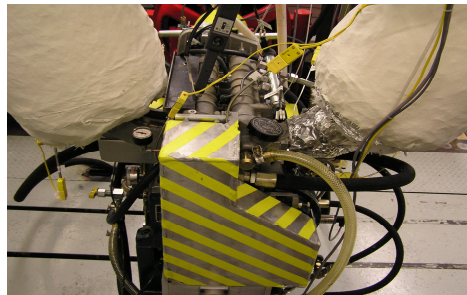
## SCE lab (June '06)



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## SCE

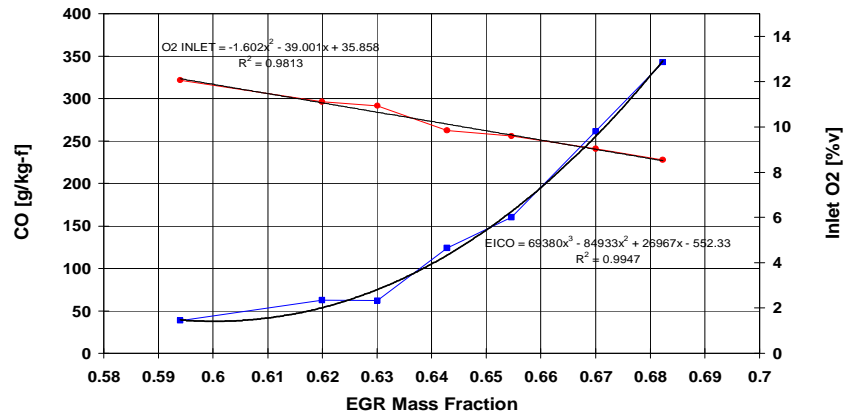
- 16.5 Compression Ratio
- 4 valves/cylinder
- 0.477 liters Displacement
- 90.2 mm Stroke, 84 mm Bore
- Common-Rail 1600 Bar Injectors flow 440mm<sup>3</sup>/30 sec., included angle 155 to 143 °
- Bosch CRIP2-MI Injectors  
CP3 Injection Pump  
EDC16 ECU



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## Experimental CO and inlet O<sub>2</sub>

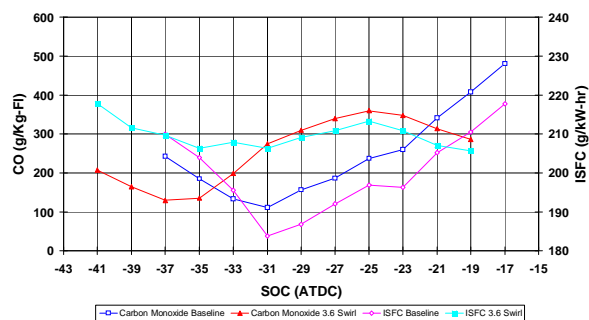


5 Bar IMEP, 2000 rpm

## For CO oxidation both temperature and mixing are important

- An optimal swirl ratio exists at which the lowest CO emission and best fuel conversion efficiency are observed likely due to mixing processes occurring after the premixed burn period.
- CO emission generally exhibits a rapid decrease from the maximum as SOI is advanced, particularly at the highest swirl ratios.

F4 Comrae 161 kPa Boost, 5 Bar Equiv IMEP, 65 C, 860 Bar Rail, 9.5 % O<sub>2</sub>, 2.2 and 3.6 Swirl



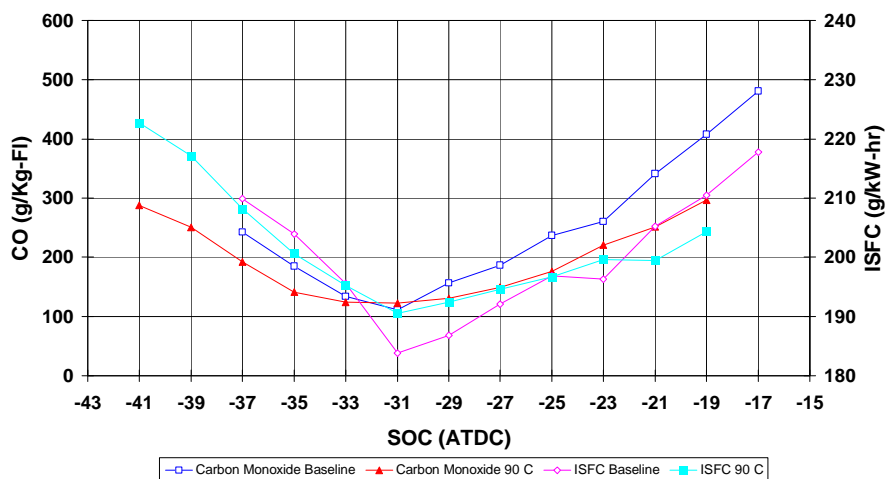
Recent data corroborates data of Miles et al. SAE 2006-01-0197

# Impact of Injection Rail Pressure

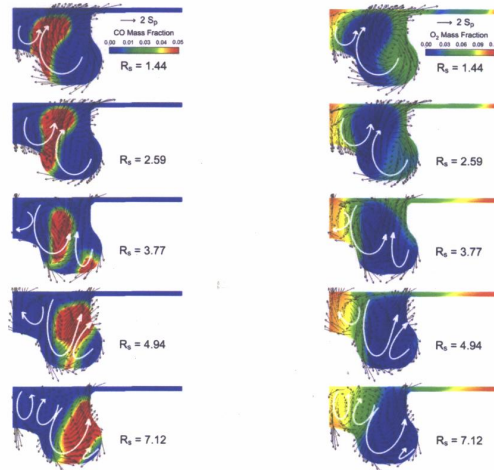
- A similar minimum in CO and ISFC is observed at different injection rail pressures
- The location of this minimum shifts with changes in rail pressure



F4 Compare 161 kPa Boost, 5 Bar Equiv IMEP, 65 C & 90 C, 860 Bar Rail, 9.5 % O<sub>2</sub>, 2.2 Swirl



## Fluid Mechanics and Chemical Kinetics are Critical!



Could it be significantly altered by fuel dependencies ?

SAE 2006-01-0197

Figure 6 The mean  $r-z$  plane flow structure and the spatial distribution of CO at 3 CAD for various swirl ratios, as predicted by numerical simulation. SOI = -22.25 CAD, cut-plane 22° down-swirl from the fuel jet

Figure 7 Changes in the spatial distribution of O<sub>2</sub> at 3 CAD for various swirl ratios, as predicted by numerical simulation. SOI = -22.25 CAD, cut-plane 22° down-swirl from the fuel jet

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## Initial Observations

- Experimental approach using intake O<sub>2</sub>% rather than EGR level works well
- UHC/CO/ISFC “sweet spot” found with this engine
- Sweet spot still observed as parameters were varied
- Mixing related control parameters (Prail, swirl) show strong effects as observed at Sandia

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